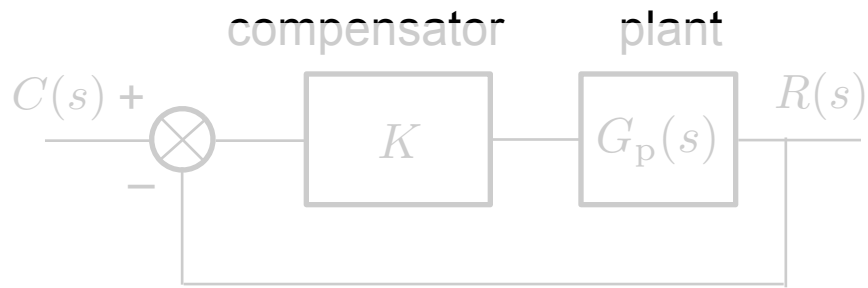



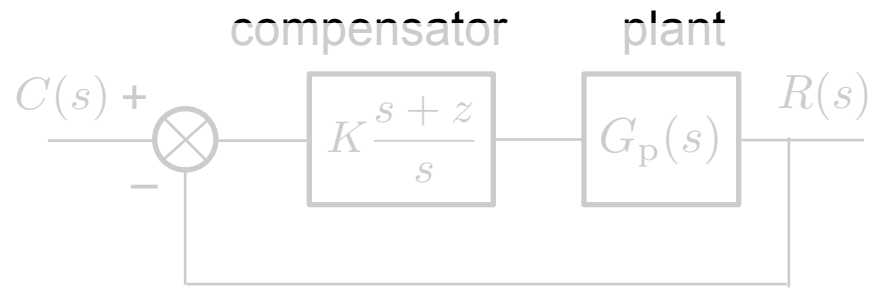


Classification of feedback compensators







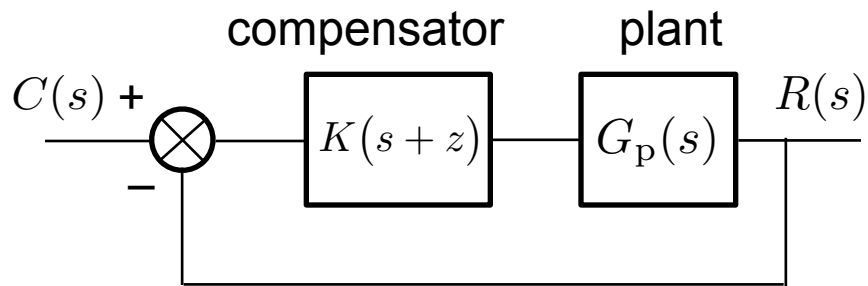
Proportional (P)

-  can meet *one* transient specification (e.g. rise time)
-  other specifications (e.g. overshoot) unmet
-  steady state error







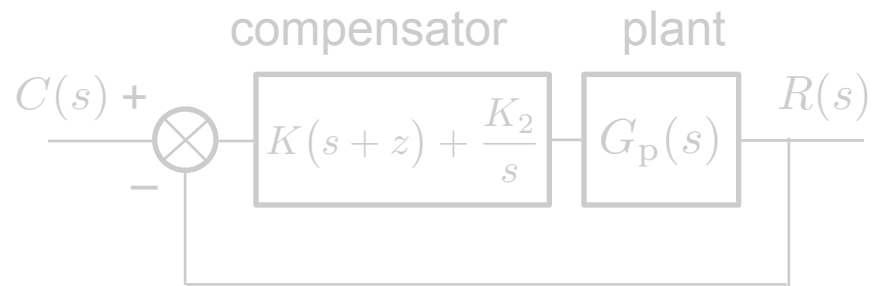
Proportional-Integral (PI)

-  can meet *one* specification (e.g. rise time)
-  zero steady-state error
-  other specifications (e.g. overshoot) unmet
-  decreases stability (RL moves to the right)





Proportional-Derivative (PD)

-  speeds up response
-  improves stability (RL moves to the left)
-  may worsen steady state error
-  noisy

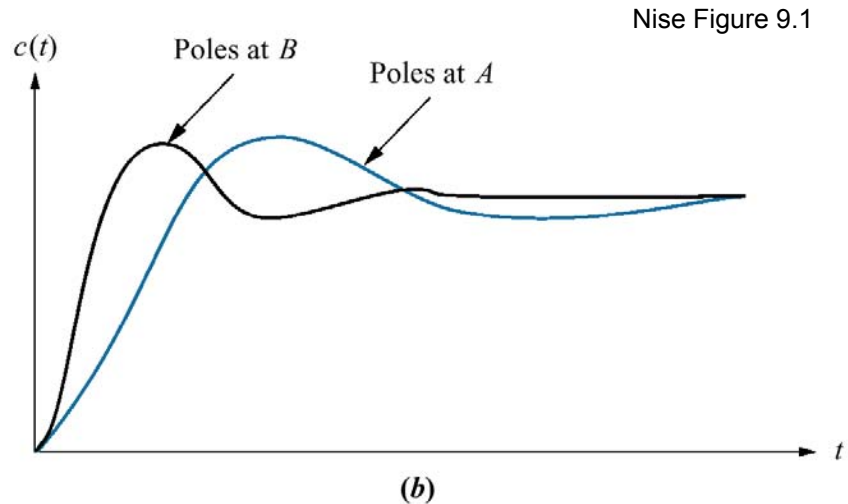
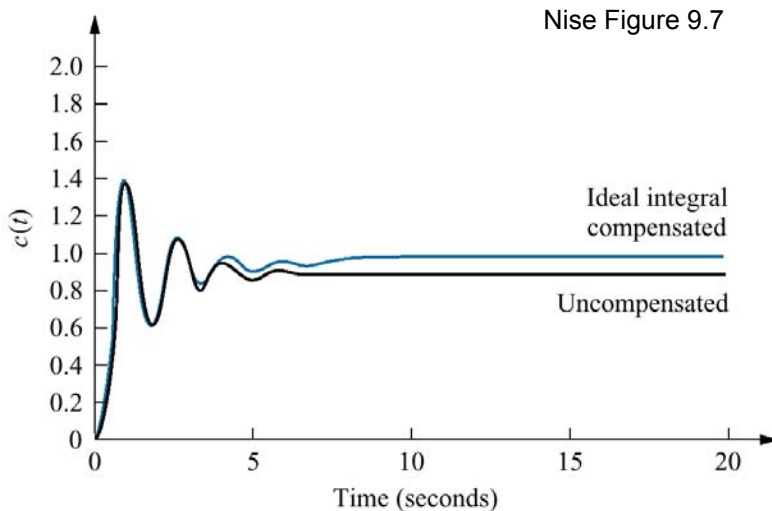


Proportional-Integral-Derivative (PID)

-  complete transient specification (rise time & overshoot)
-  no steady-state error

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Compensator rules of thumb



Integral action

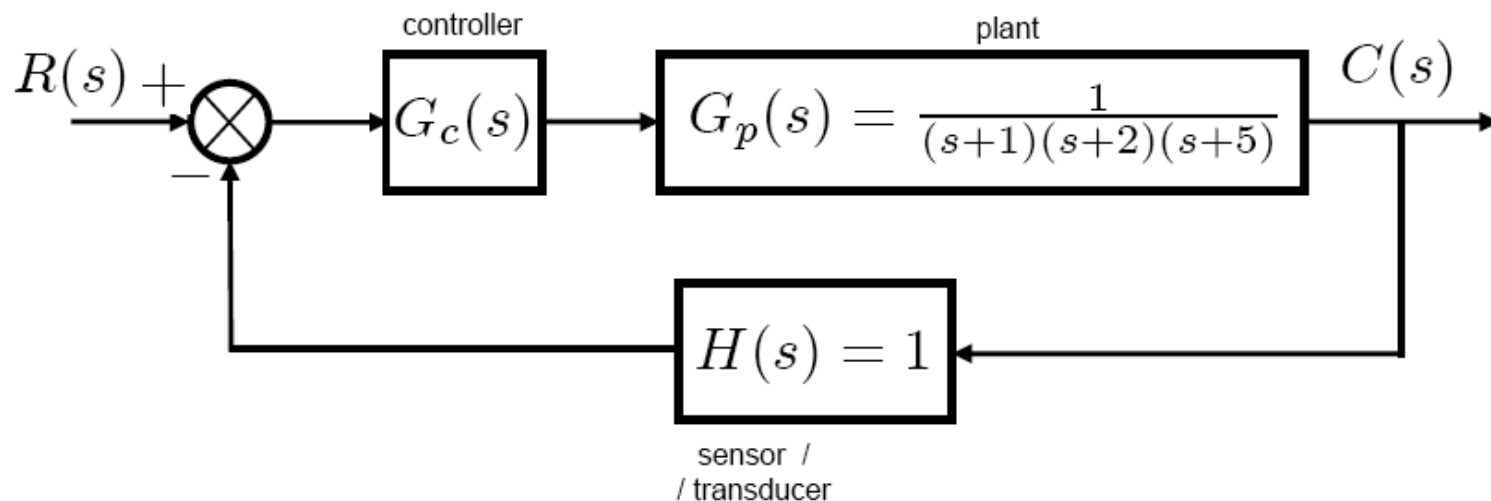
- eliminates steady-state error; but,
- by itself, the integrator slows down the response;
 - therefore, a zero (derivative action) speeds the response back up to match the response speed of the uncompensated system

Derivative action

- speeds up the transient response;
- it *may* also improve the steady-state error; but
- differentiation is a **noisy** process
 - (we will deal with this later in two ways: the lead compensator and the PID controller)

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Example



We wish to speed up the system response while maintaining $\zeta = 0.4 \Leftrightarrow \%OS \approx 25.4\%$.

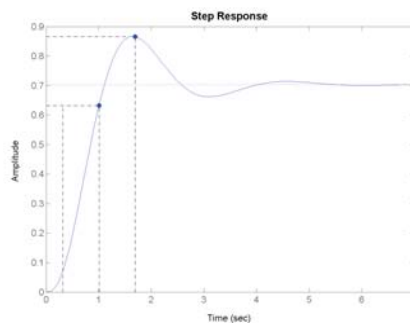
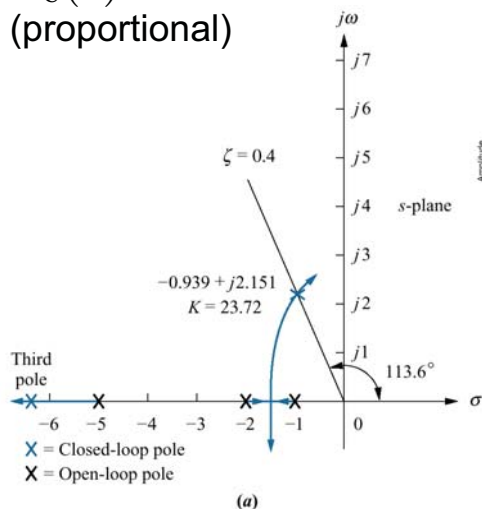
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Evaluating different PD controllers

Nise Figure 9.15

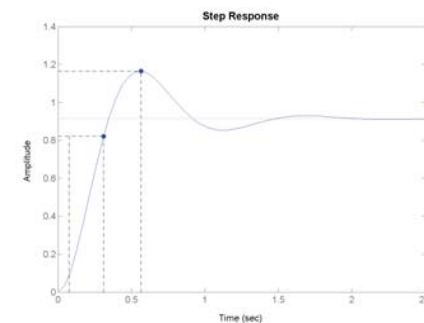
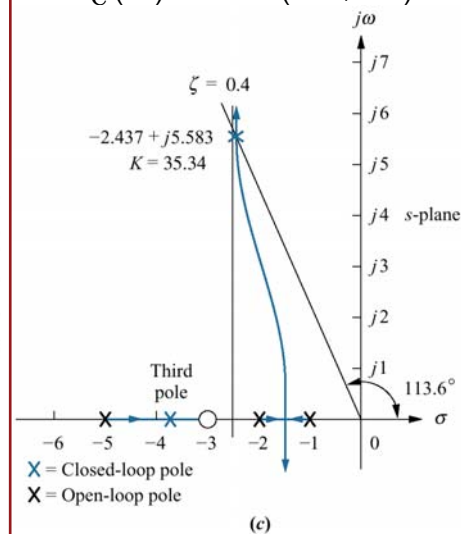
$$G_c(s) = K$$

(proportional)

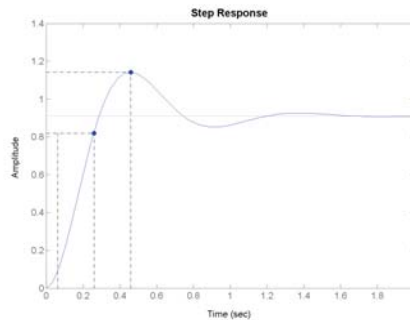
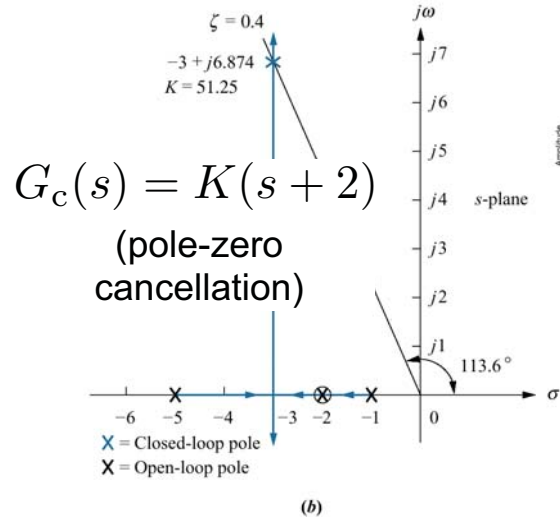


$K=23.7$
 $\%OS=23.2$
 $T_r=0.688$ sec

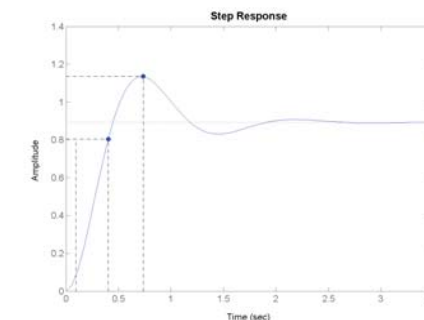
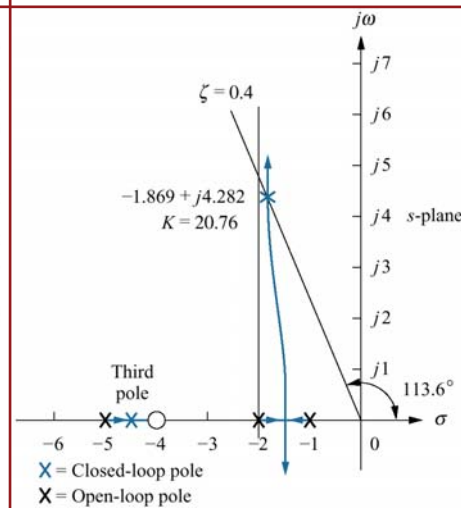
$$G_c(s) = K(s + 3)$$



$K=35.3$
 $\%OS=27.5$
 $T_r=0.236$ sec



$K=51.4$
 $\%OS=25.4$
 $T_r=0.197$ sec

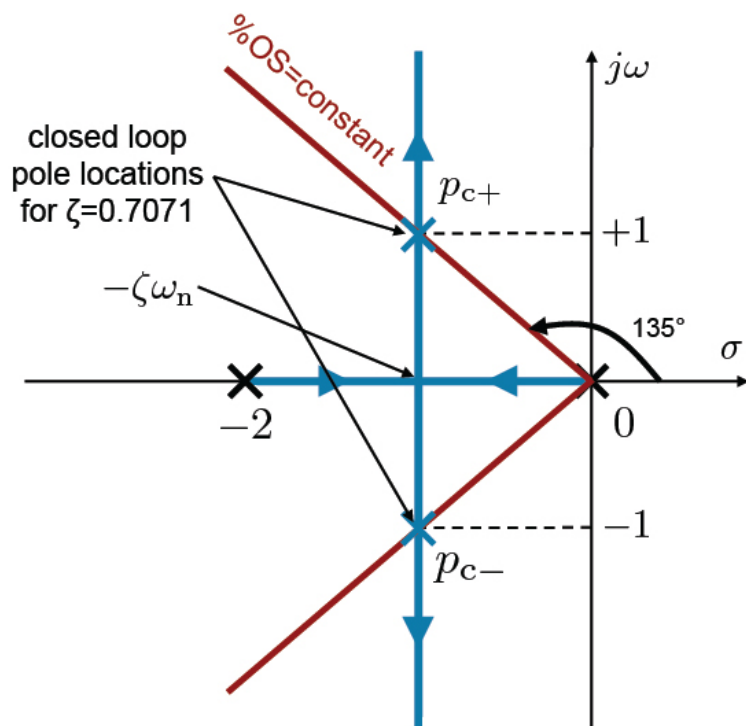
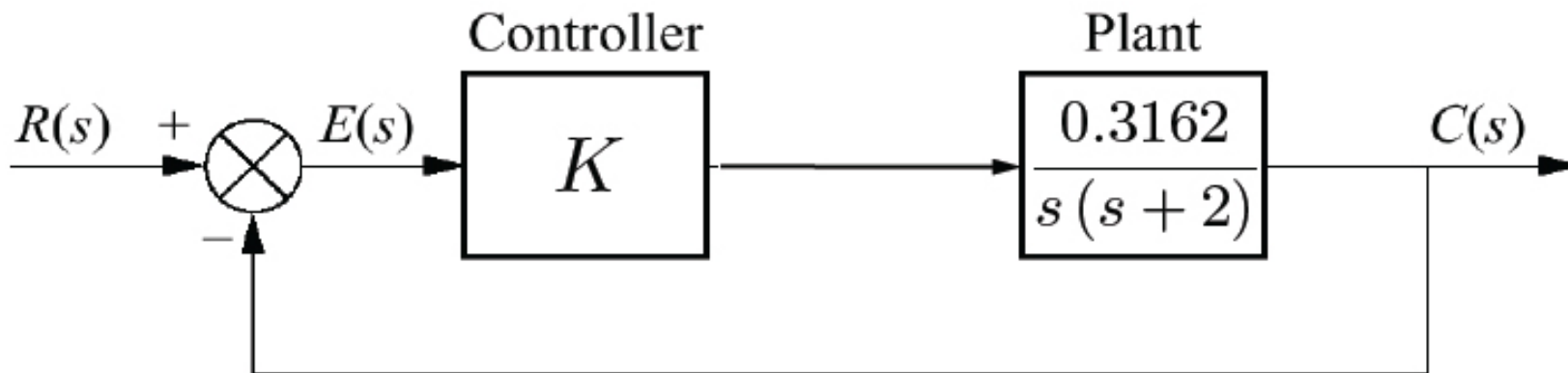


$K=20.86$
 $\%OS=27.2$
 $T_r=0.305$ sec

$$G_c(s) = K(s + 4)$$

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Example: speeding up the response



The closed-loop poles shown here are with proportional control, designed for

$$\zeta = 1/\sqrt{2} = 0.7071 \Leftrightarrow \%OS = 4.32\%.$$

We found that the overshoot target is achieved with proportional gain $K = 6.325$.

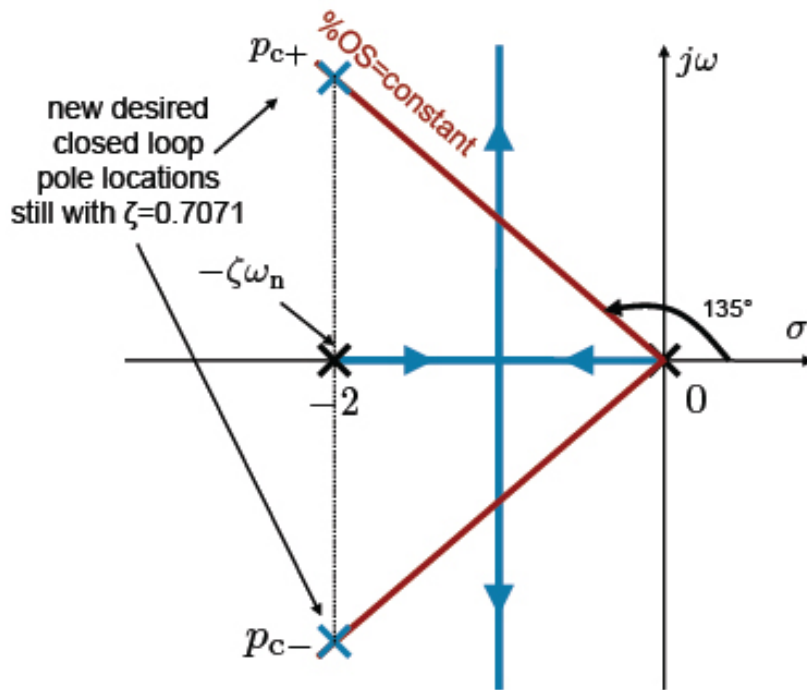
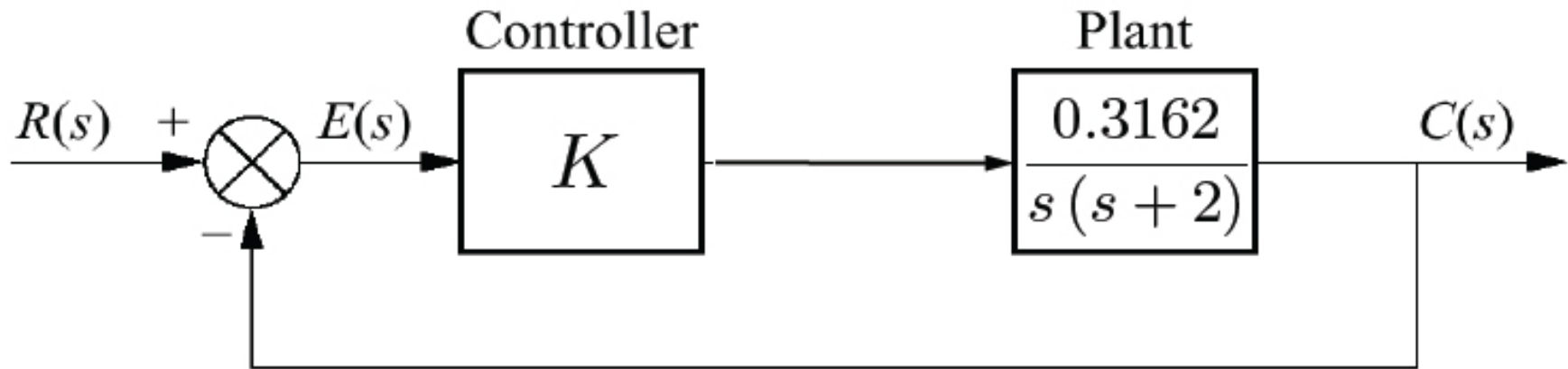
From the root locus we can see that for this value of gain, the settling time is

$$T_s \approx \frac{4}{\zeta\omega_n} = \frac{4}{1} = 4 \text{ sec.}$$

How can we “speed up” the system to $T_s = 2$ sec while maintaining the same $\%OS$ value?

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Example: speeding up the response



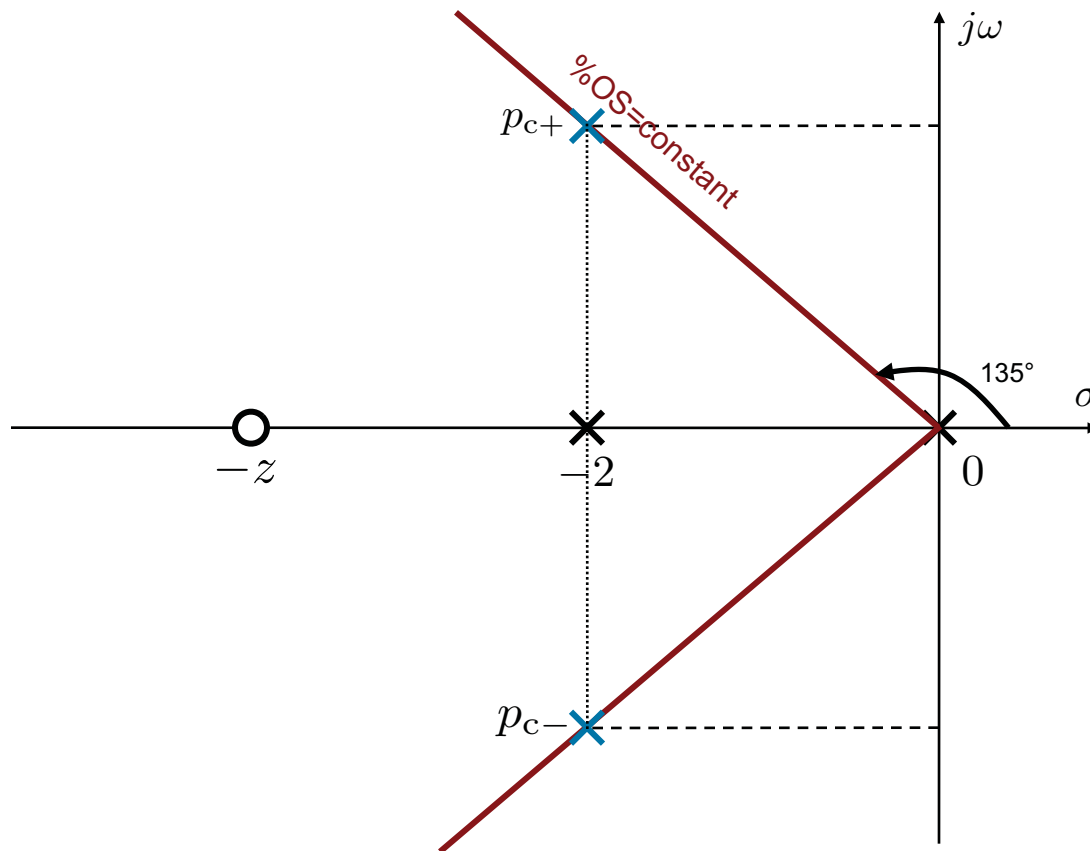
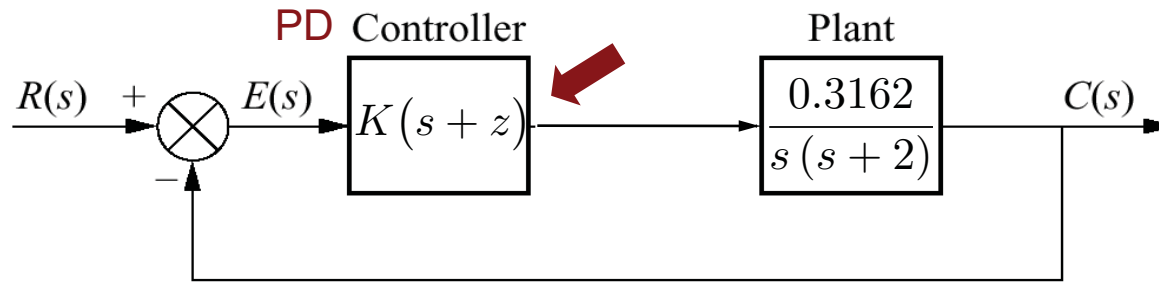
From the shorter settling time requirement, we have

$$T_s \approx \frac{4}{\zeta\omega_n} = 2 \Rightarrow \zeta\omega_n = 2.$$

Moreover, to maintain the same %OS, the poles must be located on the $\zeta = 0.707$ line. The new desired pole locations are shown on the left. Unfortunately, **they do not belong** to the uncompensated root locus. To achieve the desired poles, we propose to use a proportional-derivative (PD) compensator.

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Example: speeding up the response



The new compensated system is represented on the left on the s -plane. The desired pole locations are

$$-p_{c\pm} = -2 \pm j2.$$

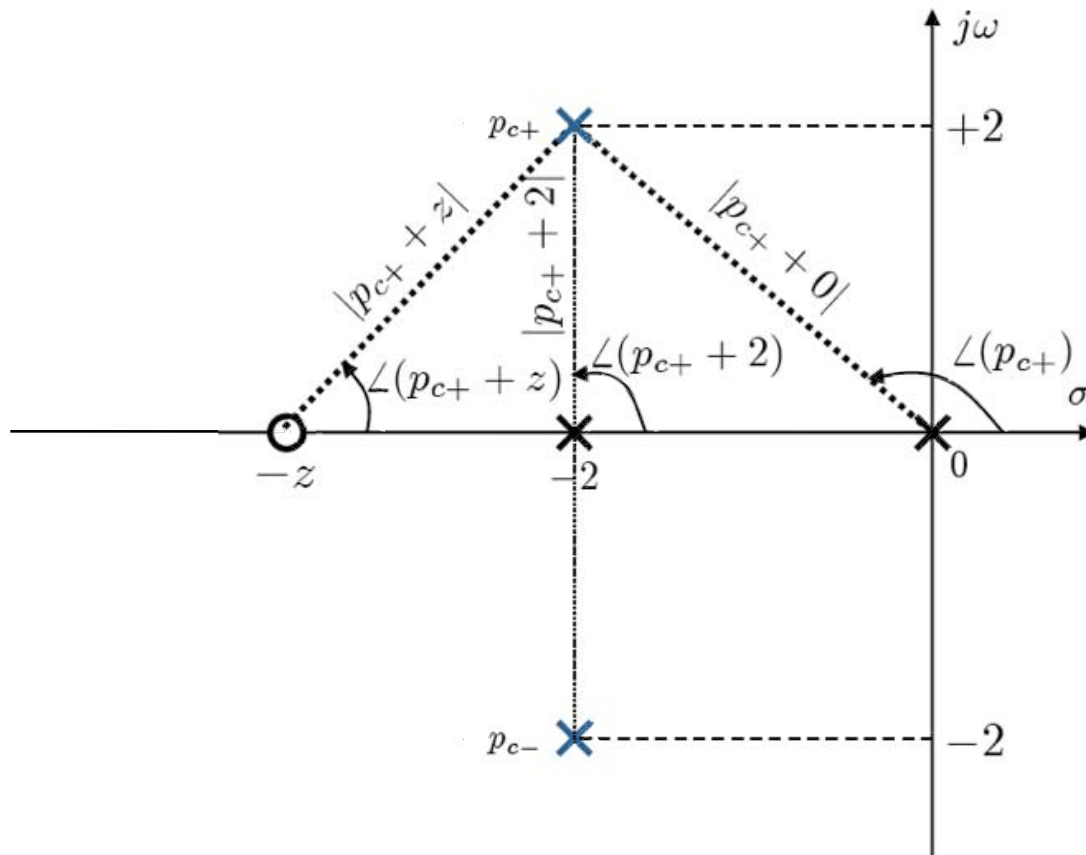
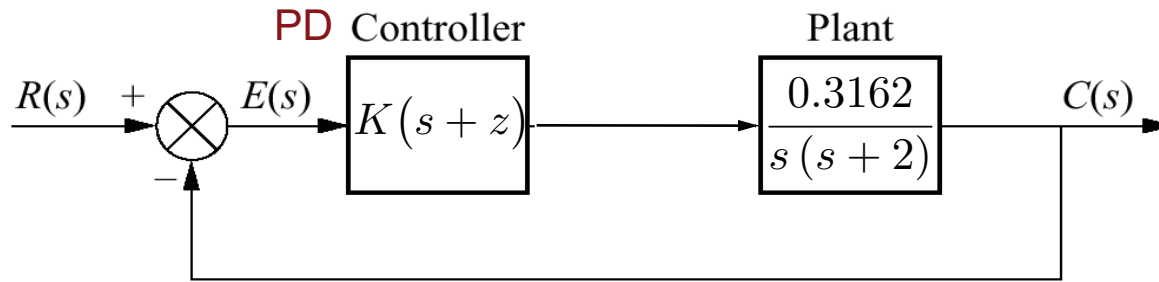
The PD compensator (see the block diagram above) contributes a zero at $-z$.

We now must answer the question: where should the zero be such that the desired poles belong on the root locus?

The process of placing z is the “design of the PD compensator.”

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Example: speeding up the response



If the new $p_{c\pm}$ are to belong to the root locus, the phase contributions from the open-loop poles must add up to $(2m + 1)\pi$, where m is an integer.

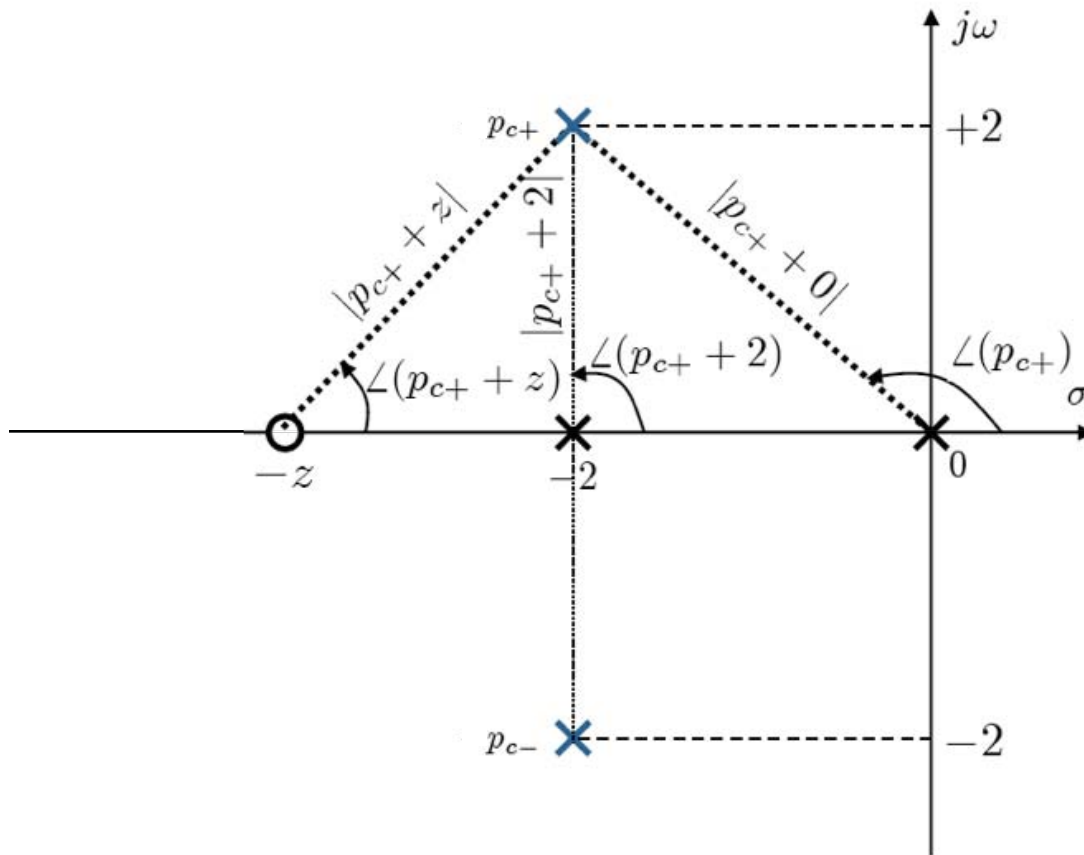
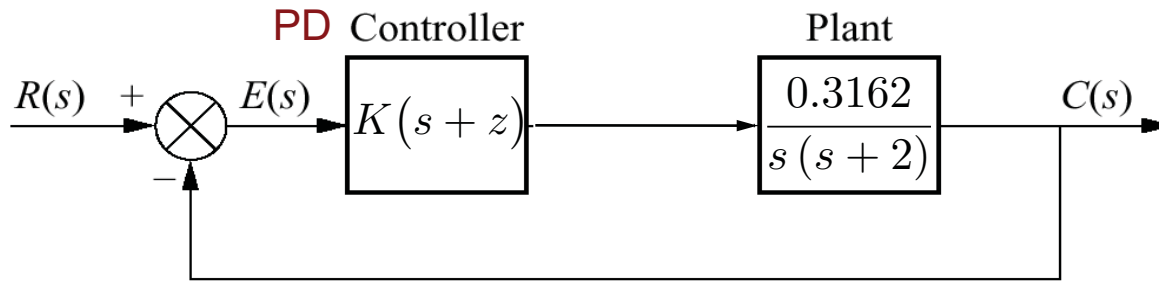
$$\begin{aligned} \angle(p_{c+} + z) - \angle(p_{c+} + 2) - \\ - \angle(p_{c+}) = (2m + 1)\pi. \end{aligned}$$

From the geometry,

$$\begin{aligned} \angle(p_{c+} + 2) &= \pi/2 \quad (90^\circ) \\ \angle(p_{c+}) &= 3\pi/4 \quad (135^\circ), \\ \Rightarrow \angle(p_{c+} + z) &= \pi/4 \quad (45^\circ), \end{aligned}$$

which places the zero at $-z = -4$.

Example: speeding up the response



Now that we have placed the zero at $-z = -4$, we can also compute the gain required to reach the desired closed-loop poles at $p_{c\pm} = 2 \pm j2$:

$$|G_c(s)| |G_p(s)| = 1, \quad \text{where}$$

$$G_c(s) = K(s + 4)$$

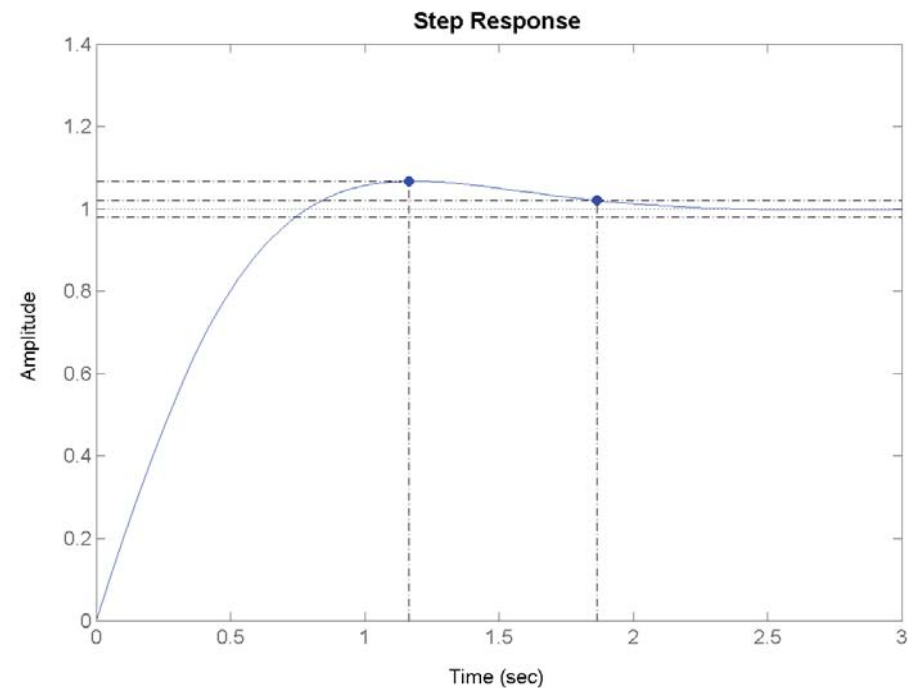
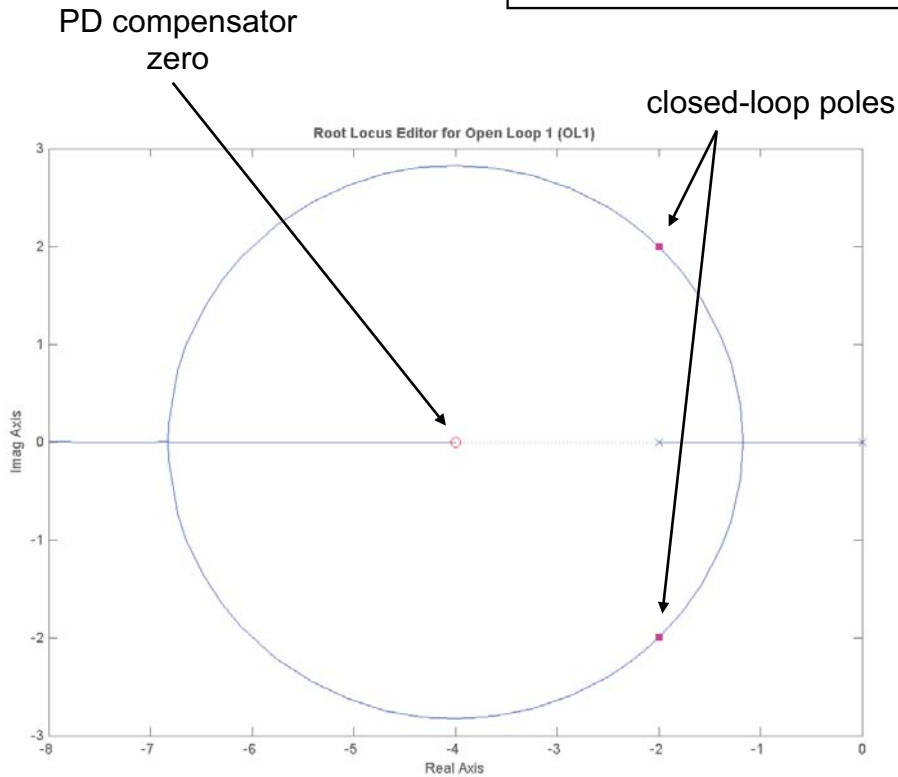
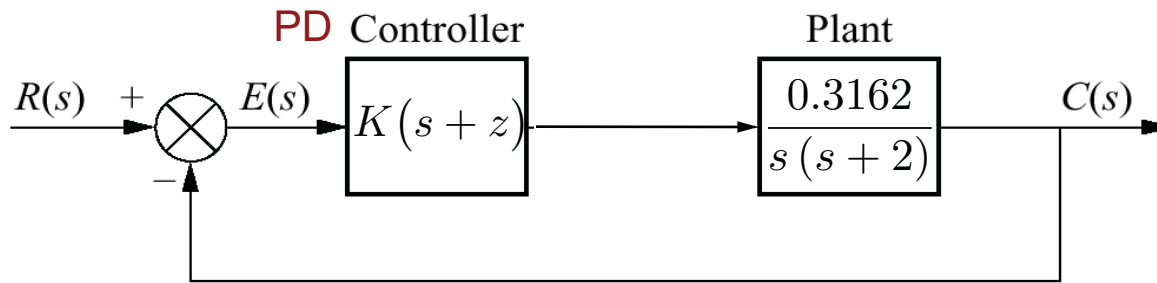
$$G_p(s) = \frac{0.3162}{s(s + 2)}$$

For the pole of interest at $s = -p_{c\pm}$,

$$\begin{aligned} \Rightarrow K &= \frac{|p_{c+}| |p_{c+} + 2|}{0.3162 |p_{c+} + 4|} \\ &= \frac{(2\sqrt{2}) \times 2}{0.3162 \times (2\sqrt{2})} = 6.325. \end{aligned}$$

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Example: speeding up the response



$K=6.325$ gives $\%OS=6.7$; $T_s=1.86\text{sec}$

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Experiments

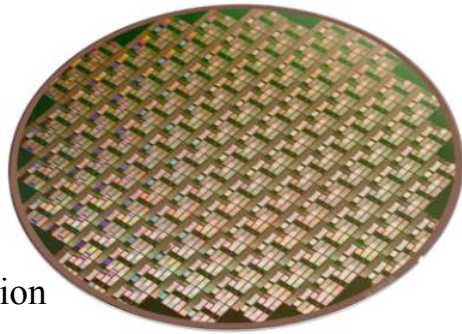


- Closed-loop position control with derivative control action:
 - **Experiment #1:** P and PD Control of Position
 - **Experiment #2:** Compare your results with a Simulink Simulation

- Deliverables:
 - Properly annotated plots showing your results
 - Comments and discussions on your observations and results

Precision Position Control Example:

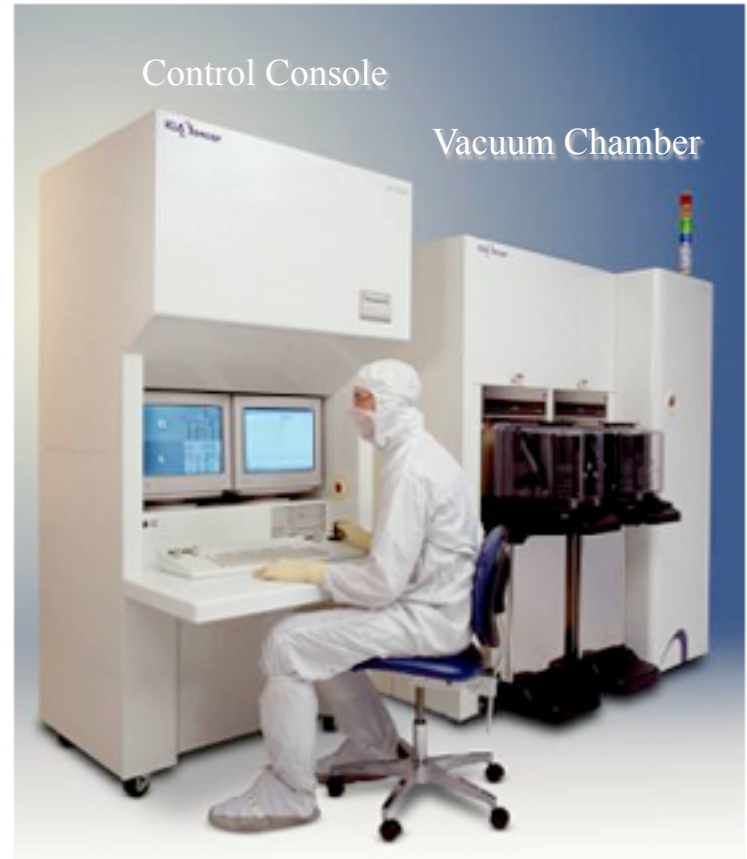
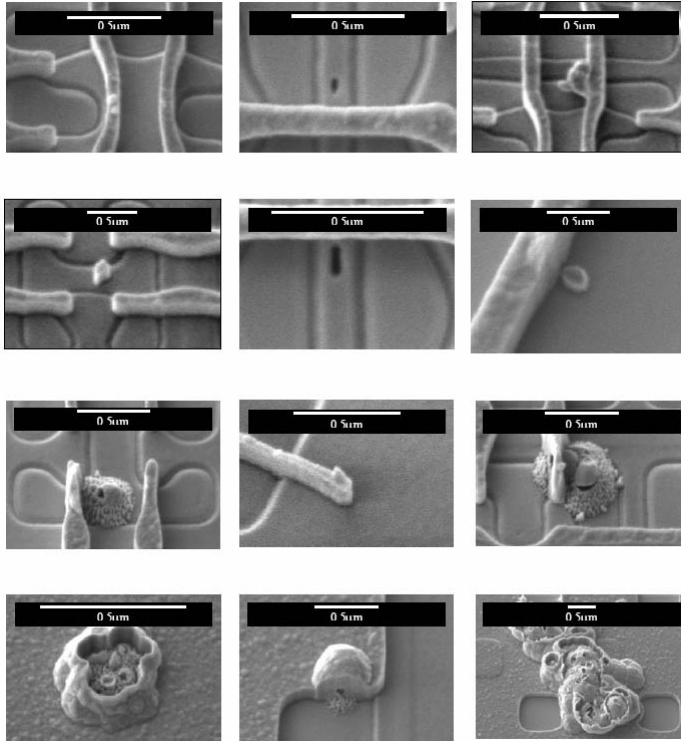
Scanning Electron Microscope (SEM) In Semiconductor Fabrication Process



300 mm diameter wafer

45 nm feature size

200,000x magnification



Control Console

Vacuum Chamber

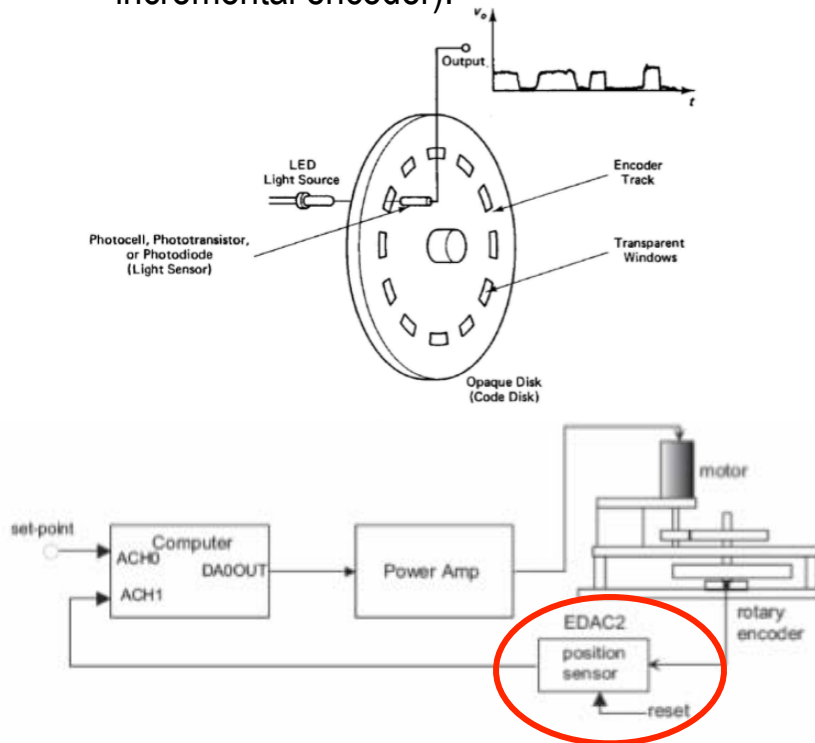
Diameter of a human hair $\approx 0.1 \text{ mm} = 1e5 \text{ nm}$

VRXLHV XONQRZ Q \$@LW KW UHVHUYHG 7KLV FROMQWLV H[FQXGHG IURP RXU &UHDVWYH
&RP P ROV @FHQVH)RUP RUH LOIRUP DWRQ VHH KWS RFZ P LVHGX KH\$ IDT IDLW XVH

Position Sensing Using Encoder and EDAC2

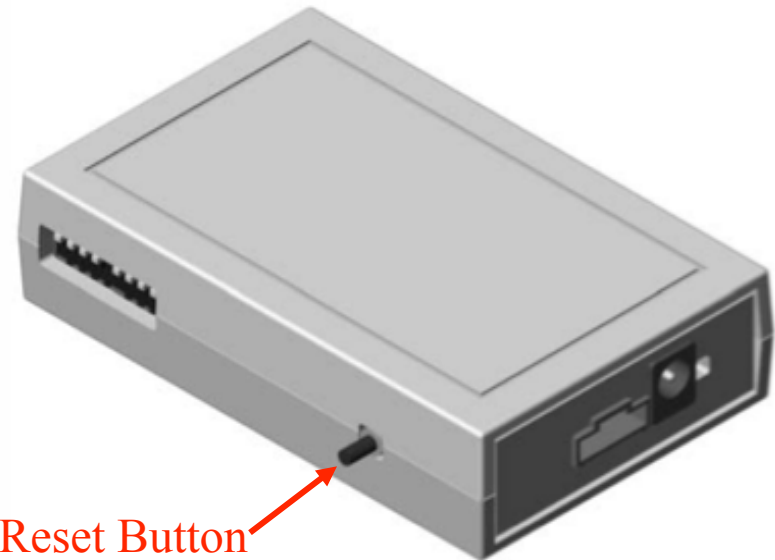
Encoder Types:

- **Incremental (Relative):** Only the relative position of the shaft is known. No absolute "0" position.
- **Absolute:** Unique code for each shaft position (e.g., by adding a reference input to an incremental encoder).



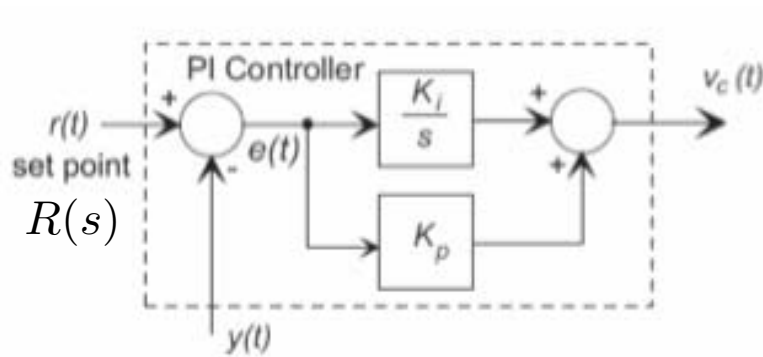
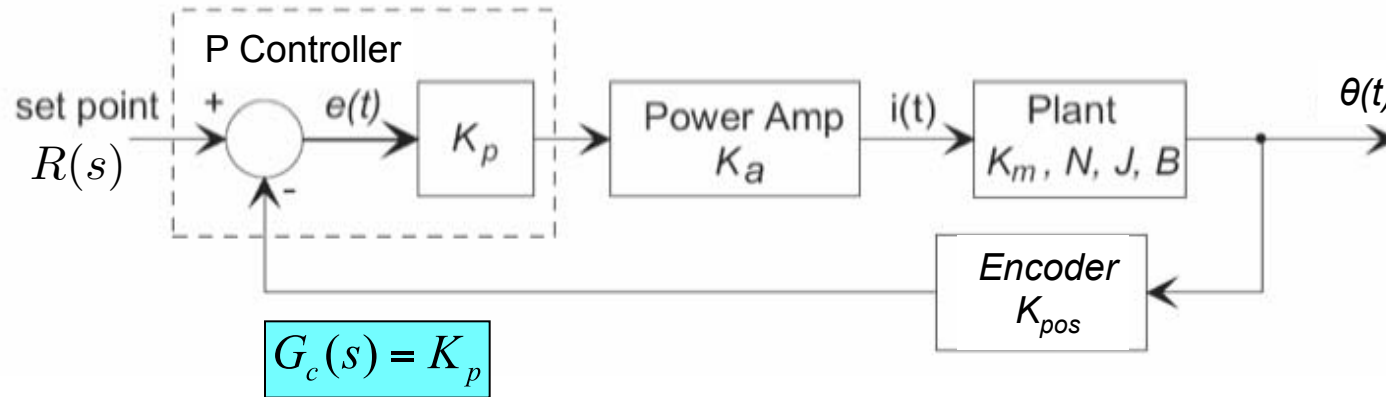
Features:

- Converts any incremental encoder into an analog position sensor
- 12 bit analog resolution
- 0 to 4.095V or 0 to 10V unipolar output voltage operation
- ± 4.095 or ± 10 V bipolar output voltage operation
- Reset can be configured to zero or mid-range voltage
- Simple DIP switch defined programming
- DIN rail mounting is available
- TTL logic level output bit to indicate direction of rotation or linear movement
- US Digital warrants its products against defects in materials and workmanship for two years. See complete warranty for details.

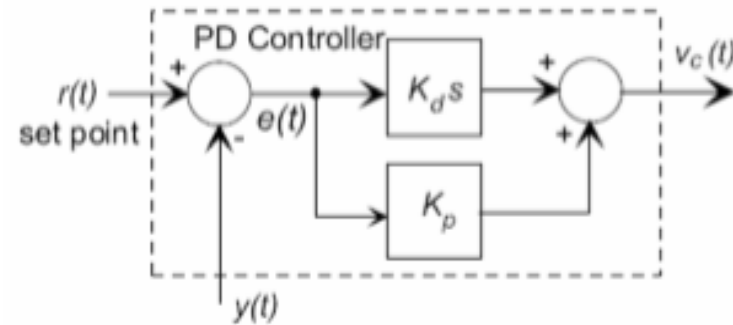


VRXLFHV XONQRZ Q \$@WJ KW UHVHUYHG 7KLV FROMQWLV H[FQXGHG IURP RXU &UHDVWVH
&RP P ROV @FHOVH) RUP RUH LOIRUP DWRQ VHH KWS RFZ P LWXGX KH\$ IDT IDLW XVH

P, PI and PD Controllers



$$G_c(s) = K_p + \frac{K_i}{s}$$



$$G_c(s) = K_p + K_d s$$

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Plant Transfer Functions



**Now we are dealing with POSITION CONTROL,
DO NOT use the velocity transfer function.**

TF voltage to velocity:

$$\frac{\Omega(s)}{V_c(s)} = \frac{K_a K_m / N}{J_{\text{eq}} s + B_{\text{eq}}}$$

position to velocity:

$$\frac{\Theta(s)}{\Omega(s)} = \frac{1}{s}$$

TF voltage to position:

$$G_p(s) = \frac{\Theta(s)}{V_c(s)} = \frac{1}{s} \cdot \frac{\Omega(s)}{V_c(s)} = \frac{K_a K_m / N}{s (J_{\text{eq}} s + B_{\text{eq}})}$$

Comparison of Closed-Loop Transfer Functions



Let $K = K_a K_m / N$

P Control:
$$G_{CL}(s) = \frac{\Theta(s)}{R(s)} = \frac{K K_p}{J s^2 + B s + K K_p K_{pos}}$$

Common form $s^2 + 2\zeta\omega_n s + \omega_n^2$

PD Control:
$$G_{CL}(s) = \frac{\Theta(s)}{R(s)} = \frac{K (K_p + K_d s)}{J s^2 + (B + K_d K K_{pos}) s + K K_p K_{pos}}$$

System Parameters



$$J \approx 0.03 \text{ N-m}^2$$

$$B \approx 0.014 \text{ N-m-s/rad}$$

$$K_a = 2.0 \text{ A/V}$$

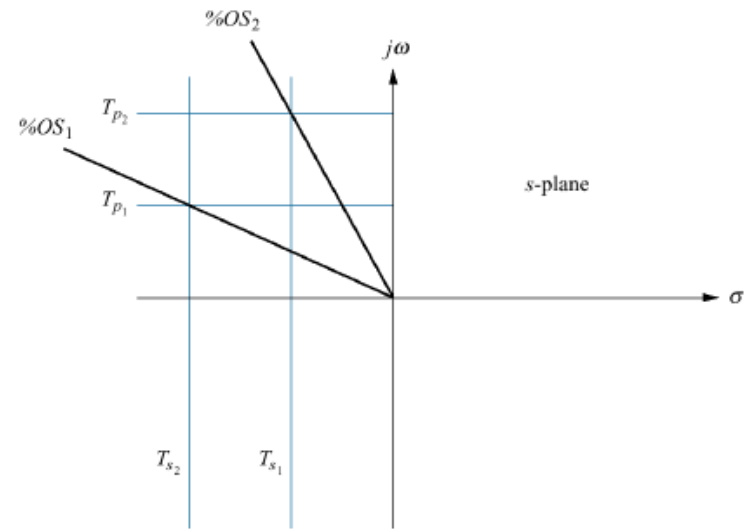
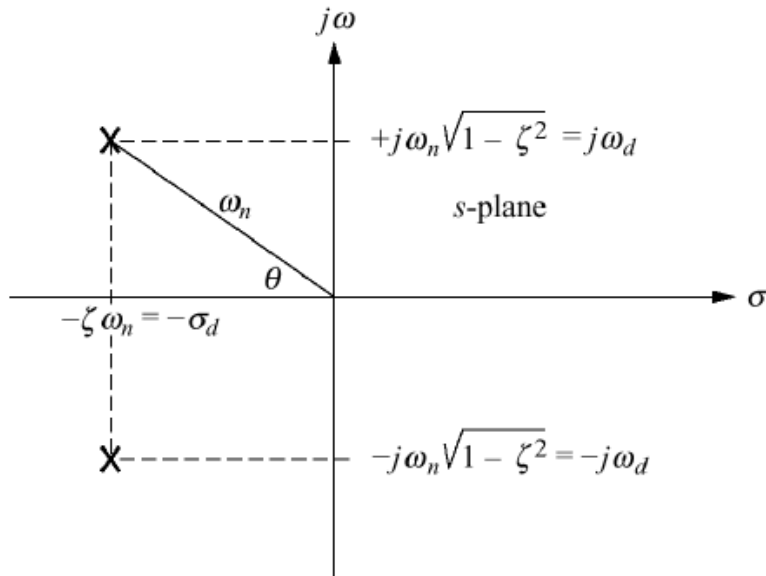
$$K_m \approx 0.0292 \text{ N-m/A}$$

$$K_t = (0.016 \frac{\text{V}}{\text{rev/min}}) (60 \frac{\text{s}}{\text{min}}) (\frac{1 \text{ rev}}{2\pi \text{ rad}}) = 0.153 \text{ V/(rad/s)}$$

$$N = \frac{44}{180} = 0.244$$

$$K_e \approx 1.5 \text{ V/rad}$$

2nd Order System Poles



$$s^2 + 2\zeta\omega_n s + \omega_n^2 = 0$$

$$s_{1,2} = -\sigma_d \pm j\omega_d = -\zeta\omega_n \pm j\omega_n\sqrt{1-\zeta^2}$$

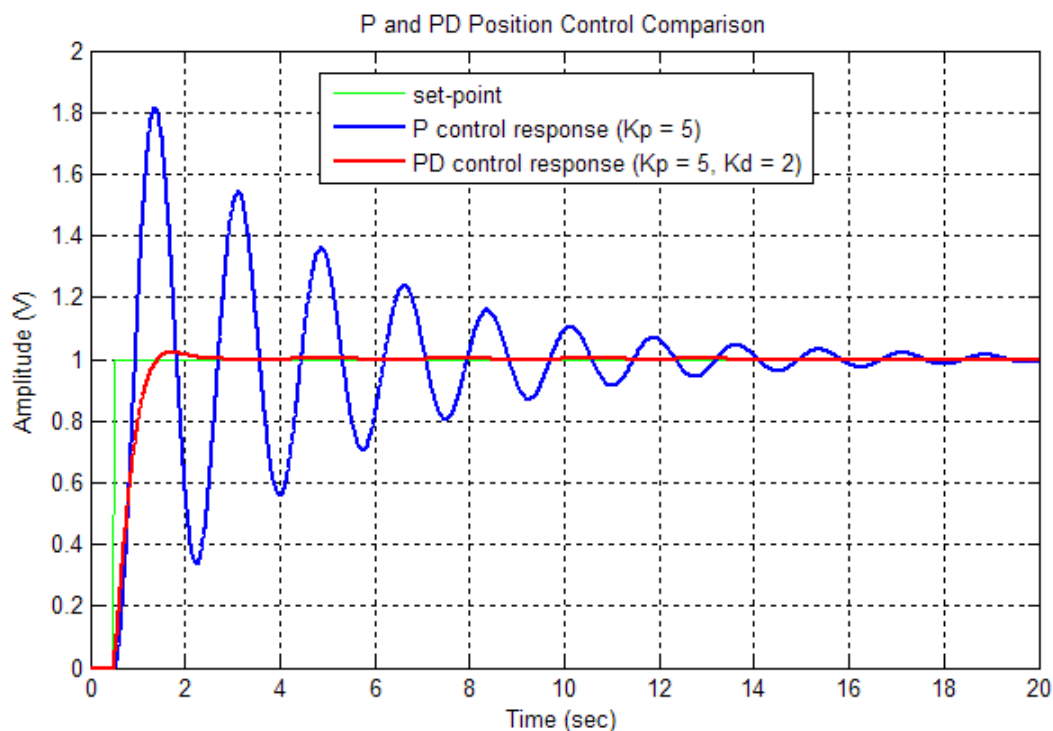
$$T_p = \frac{\pi}{\omega_n\sqrt{1-\zeta^2}}$$

$$T_s = \frac{4}{\zeta\omega_n}$$

$$\%OS = e^{-\left(\frac{\zeta\pi}{\sqrt{1-\zeta^2}}\right)} \times 100$$

Control Action Comparison

- ***P*** – improve speed but with steady-state error
- ***D*** – improve stability but sensitive to noise
- ***I*** – improve steady state error but with less stability, overshoot, longer transient, integrator windup (we will discuss *PI* and *PID* control next week)



Procedure EX1



- Connect the computer-based as before except we need to use the EDAC2 instead of ETACH2. The set-up is very similar. Install one magnet.
Important: ALWAYS RESET EDAC2 BEFORE EACH TRIAL
(The flywheel could go out of control, be ready to stop the loop at any time.)
- Before starting the experiment, spin the flywheel so that the position mark on the flywheel is observable and define an initial position.
- Set the function generator to output a DC signal with 1.0 V offset. Set K_p to 2, 3, 4 and start experiment. Record a transient response for each case.
- Spin the wheel back to its initial position. Set K_p to 2 and K_d to 1. Repeat the experiment. Try two other (reasonable) combinations of K_p and K_d control parameters. Compare P and PD control results.
- Select your favorite combination of PD control parameters, run experiment as before after the plant reaches steady state, “continuously” change your DC offset by pressing the “up” or “down” button; observe controlling of the plant.

Procedure EXP2



- Define a Matlab SISO model to represent the flywheel with voltage as input and position as output. Simulate a P control with $K_p = 2$.
- Change your controller design to PD. Choose 2 combinations of K_p and K_d values from EXP1 and run simulation.
- Compare your simulation with results from EXP1 and comment on agreement or discrepancy between theory and experiment.

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2.04A Systems and Controls
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